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## Mechanical analysis for the Monolith Vessel

**Consorcio ESS-BILBAO & Instituto de Fusión Nuclear**

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# Introduction

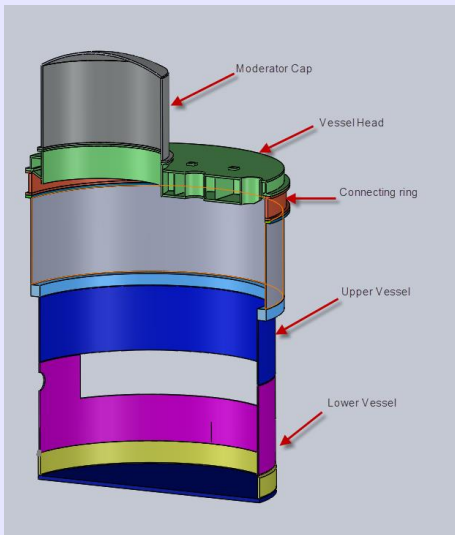
# ESS-BILBAO Consortium

## Role and functions

- The Spanish Government has taken the decision that ESS-BILBAO will be the only contractor from Spain to ESS project.
- ESS-BILBAO has no independent research activities neither teaching responsibilities outside ESS project.
- Staff of 60 scientific & engineers and the possibility to hire extra staff.
- On January 2015, ESS-Bilbao was chosen as ESS partner for the Monolith Vessel.
- On September 2015, ESS-Bilbao and ESS-ERIC organized the KO meeting in Madrid.
- On November 2015, ESS-ERIC completed the PDR and ESS-Bilbao started the design phase.

# Introduction

## Monolith Vessel components



# Planing and Schedule

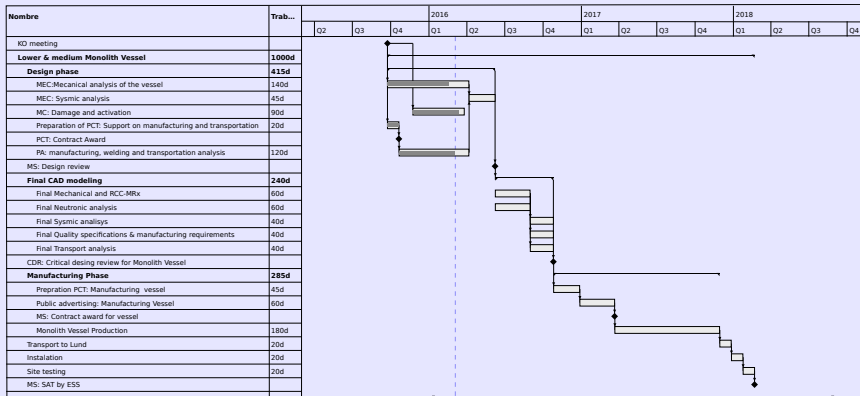
# Planing and Schedule

## Modifications based on the actual scenario

- We have divided the project in two sub projects with different schedule and SAT: “Lower & Medium Vessel” and “Head of the Vessel & Connection Rings”.
- The information required for the design process for the “Lower & Medium Vessel” is mainly on place so the design is progressing on schedule.
- Regarding “Head of the Vessel & Connection Rings” additional clarifications about the number, size and position of the different penetrations have to be provided by the end of the month.
- A clear definition of the vacuum requirements is critical to continue the manufacturing review and the definition of the flanges and joints.

# Planing and Schedule

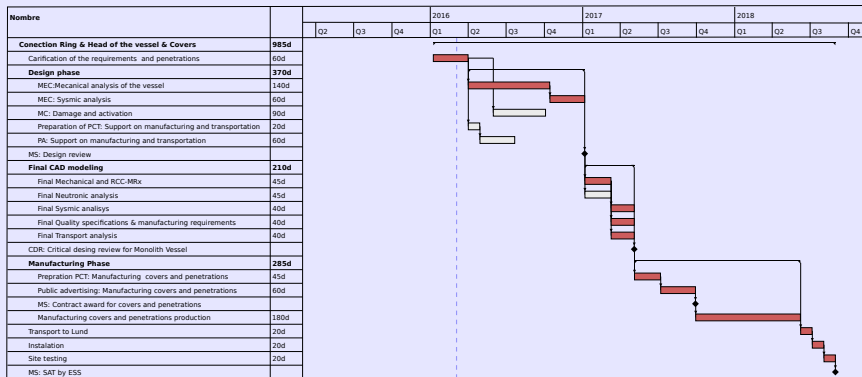
## Schedule





# Planing and Schedule

## Schedule



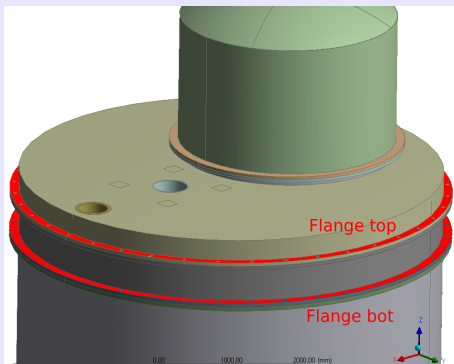
# Radiation Damage Analysis

# Radiation Damage Analysis

## Flanges and seals

The proposed solution for the joint between Connection ring, Head of the Vessel and Medium vessel is based on a bolted flange. An evaluation of the radiation level is need to clarify the type of seal (elastomer or metal) . If a metallic seal is needed the radiation level can be consider negligible ( $\ll 1E-3$  dpa).

## MCNPX model

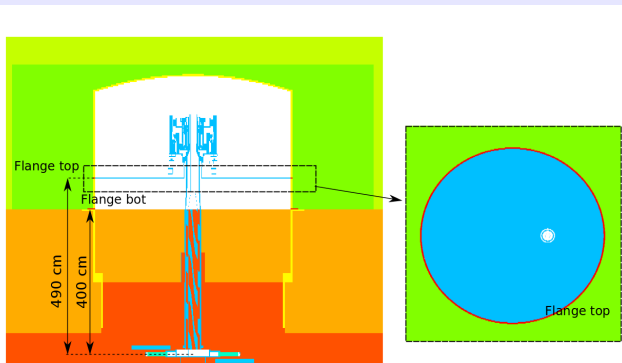


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## MCNPX model



# Radiation Damage Analysis

## Flanges and seals

The proposed solution for the joint between Connection ring, Head of the Vessel and Medium vessel is based on a bolted flange. An evaluation of the radiation level is need to clarify the type of seal (elastomer or metal) . Considering 40 years of operation, the damage levels are in the range of standard vacuum elastomer (<800 Sv).

## Dose rate for 5000 h of operation per year (neutrons & photons).

Element	Gy/year (Steel)	Sv/year	DPA/year
Top Flange	2.5	24.3	$6.2 \cdot 10^{-10}$
Bottom Flange	0.8	9.6	$1.8 \cdot 10^{-10}$

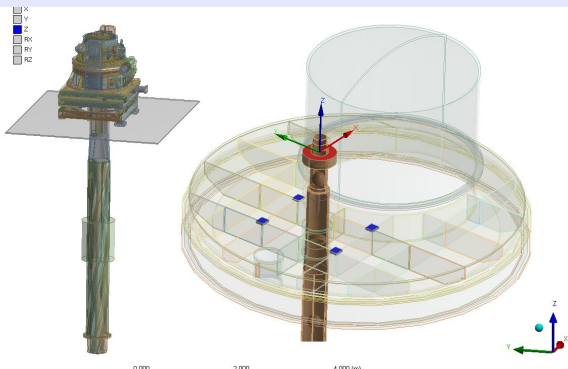
# Mechanical analysis

# Mechanical analysis

## Main loads and boundary conditions

A FEM model has been configured for the mechanical analysis. The Target has been considered as a shaft with point masses (mass and inertia for internal shielding and Target wheel) connected to the vessel by springs ( $1E9 \text{ N} \cdot \text{m}^{-1}$ ) and a joint connection. This simplifications keeps the main resonance modes for the shaft.

## Geometrical model for the monolith vessel

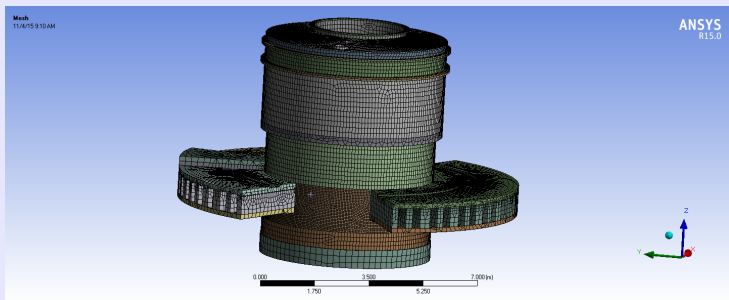


# Mechanical analysis

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## Geometrical model for the monolith vessel





# Material selection

## Main loads and analysis

- Dead weight (Target and vessel)
- Vacuum (P-loads & buckling)
- 2.0 bars Over pressure (P- loads & bolted union analysis)
- Seismic analysis

## Based on vacuum requirements two stainless steels could be considered

- SS-316L: Better corrosion resistance properties (**Base line**)
- SS-304: Better manufacturing and welding properties.

# Mechanical analysis: Vacuum

Stress and deformations relative low compared with acceptable levels ( $\sim 20\text{-}30\text{ MPa}$ )

B: Static Structural

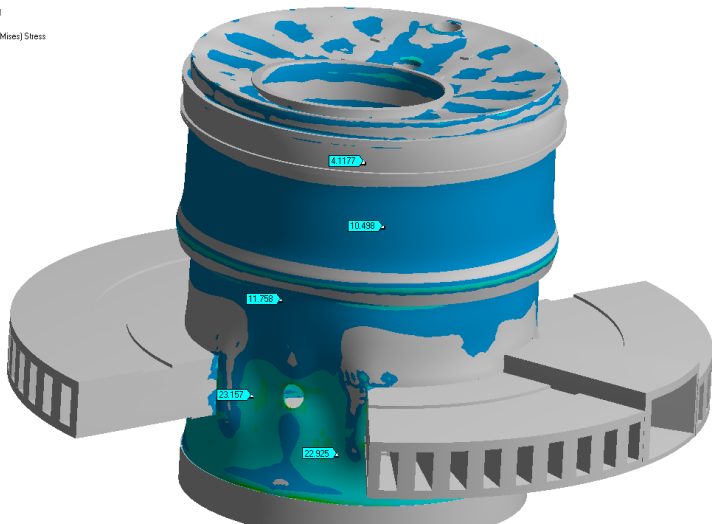
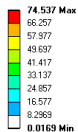
Equivalent Stress

Type: Equivalent (von-Mises) Stress

Unit: MPa

Time: 1

1/8/16 3:58 PM

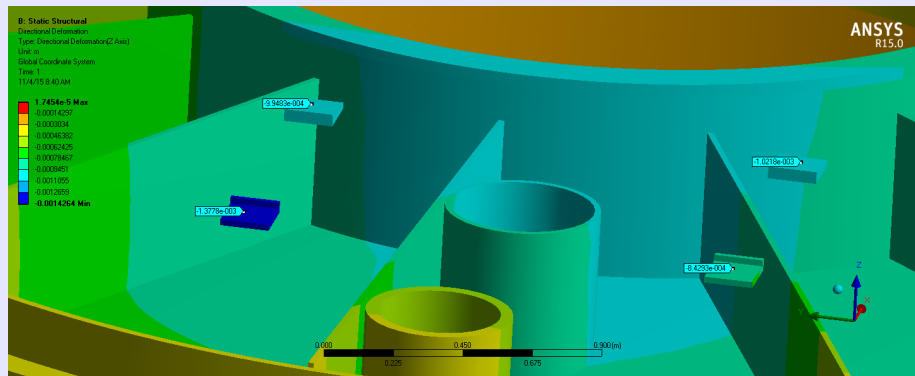


# Mechanical analysis: Vacuum

## New configuration for the vessel head stiffness

One of the most demanding design criteria is the requirement to minimize the vertical displacement of the target vessel supports. Also it is convenient to have an homogeneous displacement of the supports to keep the target balanced.

## The displacement of the drive unit supports

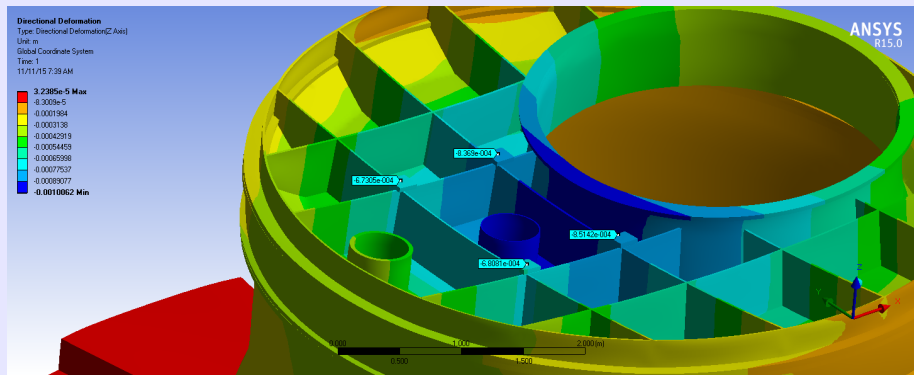


# Mechanical analysis: Vacuum

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## The displacement of the drive unit supports

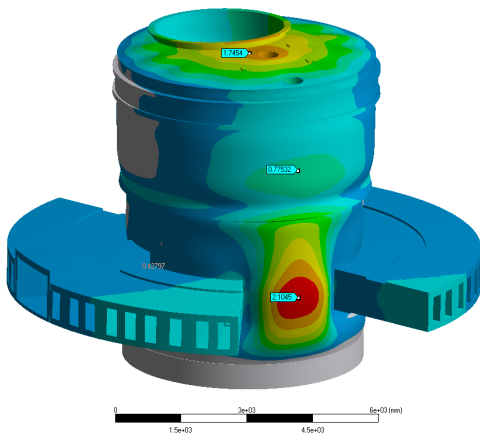
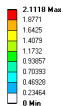


# Nominal conditions: Over pressure 2 bar

## The displacement of the drive unit supports

0: Static Structural: Overpressure

Total Deformation  
Type: Total Deformation  
Unit: mm  
Time: 1  
1/12/16 7:39 AM



# Nominal conditions: Over pressure 2 bar

## The displacement of the drive unit supports

G: Static Structural: Overpressure

Equivalent Stress

Type: Equivalent (von Mises) Stress

Unit: MPa

Time: 1

1/12/16 7:39 AM

151.25 Max

134.45

117.65

100.85

84.048

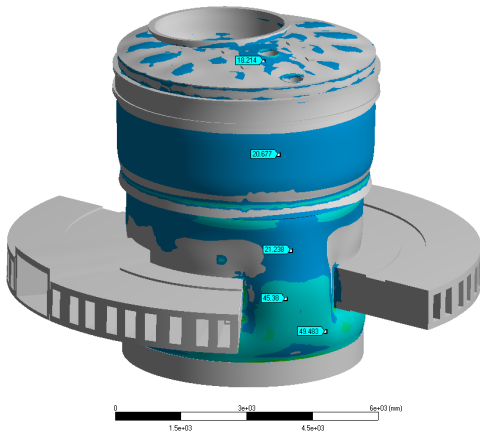
67.248

50.448

33.647

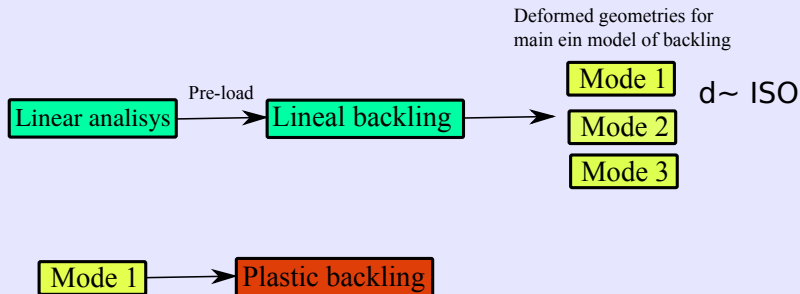
16.847

0.046277 Min



# Mechanical analysis: Buckling

Methodology based on RCC-MRx



# Mechanical analysis: Linear buckling

## Linear buckling "safety factors" for SS-316L

Mode	$\lambda_c$
1.	22.916
2.	24.221
3.	24.974
4.	25.995
5.	26.029
6.	26.141
7.	26.743
8.	26.75
9.	26.884
10.	26.896

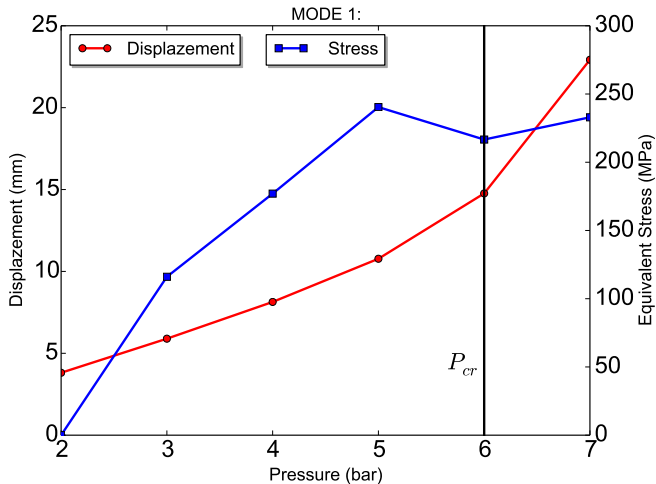
## Linear buckling

The linear buckling analysis can produce a significant error and it is not covered for all materials at RCC-MRx. We have complete the analysis for the first 30 modes.



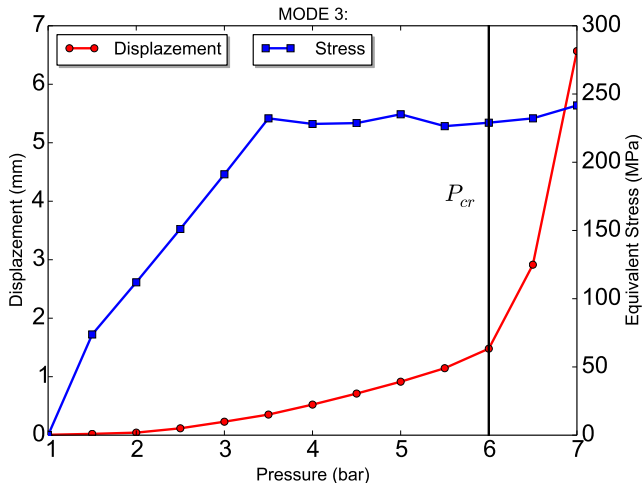
# Mechanical analysis: Plastic Buckling for SS-304

## Modes 1 and 3



# Mechanical analysis: Plastic Buckling for SS-304

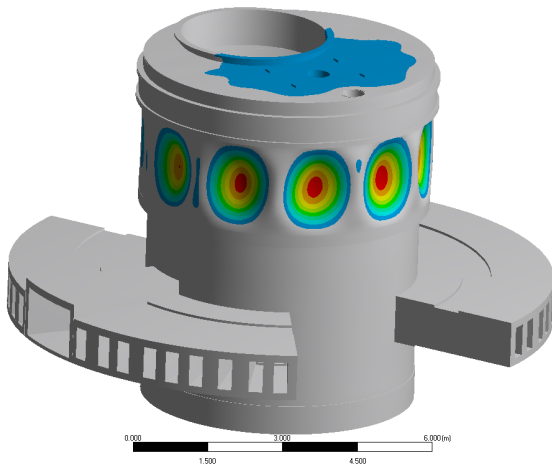
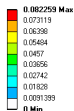
## Modes 1 and 3



# Mechanical analysis: Plastic Buckling for SS-304

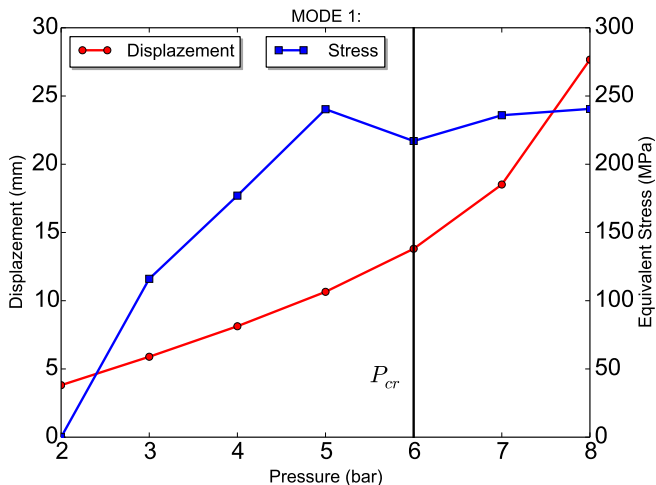
## Modes 1 and 3

C: Mode 3  
Total Deformation 2  
Type: Total Deformation  
Unit: m  
Time: 1 (Unconverged)  
1/8/16 2:53 PM



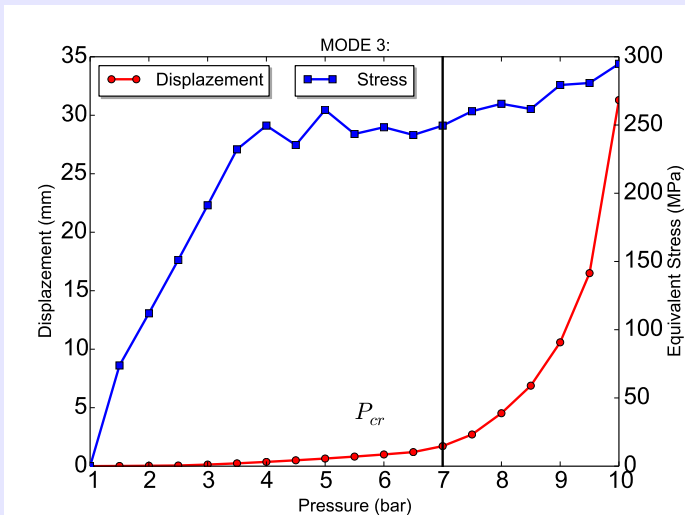
# Mechanical analysis: Plastic Buckling for SS-316L

## Modes 1 and 3



# Mechanical analysis: Plastic Buckling for SS-316L

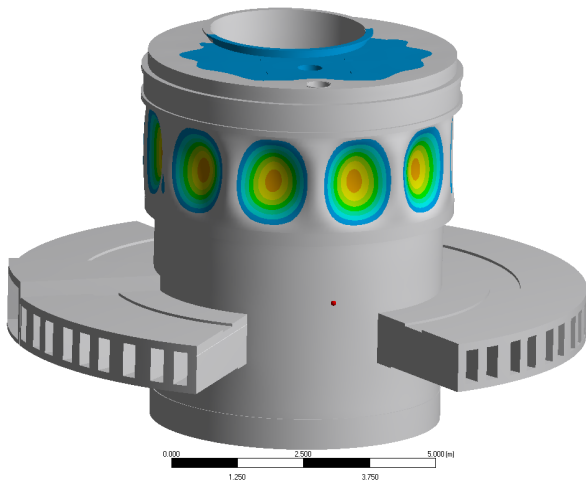
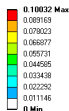
Modes 1 and 3



# Mechanical analysis: Plastic Buckling for SS-316L

## Modes 1 and 3

C: Mode 3  
Total Deformation 2  
Type: Total Deformation  
Unit: m  
Time: 1  
1/8/16 4:10 PM



# Mechanical analysis: Conclusions

## Remarks from mechanical analysis

- The Stress values are relatively far below the RCC-MRx limits so there is room for thicknesses optimization, specially on lower and medium vessel.
- The displacement in the Target supports can be limited to 2 mm
- Seismic analysis are on going.

# Manufacturing Analysis

## Review of manufacturing procedure

On December 2015, ESS-BILBAO awarded a contract to ENWESA to review the manufacturing process for the lower and medium vessel (manufacturing, testing, transportation and installation in Lund). The work should be completed in a in a couple of weeks.

## ENWESA



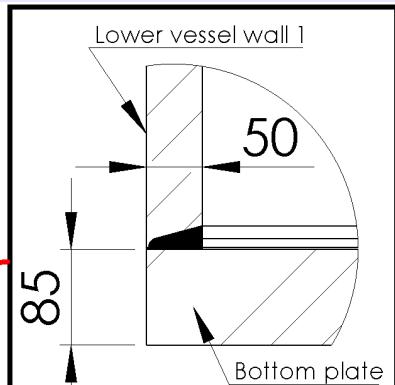
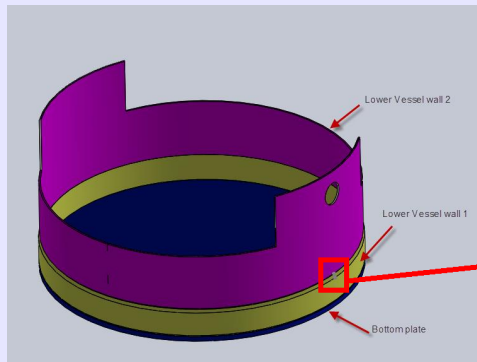


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## Welding analysis



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## Welding analysis

	III.1	fillet or T	full penetration	two sides accessible	back weld or back machining
	III.2	fillet or T	full penetration	back side inaccessible	gaseous back protection
	III.3	fillet or T	full penetration	back side inaccessible	permanent backing strip

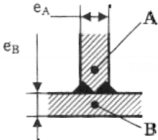
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## Welding analysis

*Table RS 7720 a : class N2<sub>Rx</sub> shell- vessel welds  
1 : examination of surface to be welded (before welding)*

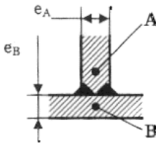
Type	Austenitic			Non destructive examination	Criteria
	Ferritic	Aluminium	Zirconium		
1. Butt welded joint (Types I.1, I.2, I.3)	ROLLED FORGED ( $e \geq 10$ mm)	X	X	RES or MAG RES	RS 7363 RS 7363 RS 7363
	( $e \geq 5$ mm)	X	X	RES	RS 7363
	CAST (all thicknesses)	X	X	RES or MAG RES	RS 7363 RS 7363
	3. Full penetration angle joint (Types III.1, III.2)				
	PART A ( $e_A \geq 10$ mm)	X	X	RES or MAG RES	RS 7363 RS 7363
	( $e_A \geq 5$ mm)	X	X	RES	RS 7363
	PART B all thicknesses	X	X	RES or MAG RES	RS 7363 RS 7363
	In addition $e_B \geq 10$ mm	X	X	US (2)	Tome 2

# Manufacturing Analysis

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## Welding analysis

Table RS 7720						
<i>a : class N2<sub>ex</sub> shell- vessel welds</i>						
<i>2 : examination during weld operations</i>						
Type	Austenitic	Ferritic	Aluminium	Zirconium	Non destructive examination	Criteria
	X	X	X	X		
	X	X	X	X		
	X	X	X	X		
1. Butt welded joint (Types I.1, I.2, I.3)	Backing run				X	RS 7363
	Each change of process (RS 7722.11)				X	RES or MAG (1) RES
3. Full penetration angle joint (Types III.1, III.2)		<p>If neither RAD nor US can be performed on completed weld, examination of root pass and examination every 3 layers during filling up of the weld (RS 7723.1)</p>			X	RS 7724.1 RS 7724.2
					X	X

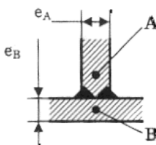
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## Welding analysis

<i>Table RS 7720</i>		<i>a : class N2<sub>Rx</sub> shell- vessel welds 3 : examination after weld operation</i>				Non destructive examination	Criteria	
Type	Austenitic				X			
	Ferritic					X		
Aluminium				X				
Zirconium				X				
1. Butt welded joint (Types I.1, I.2, I.3)	All thicknesses :				X	X	RES	RS 7724.1 RS 7724.2 RS 7724.3 RS 7724.4 (*)
	(outer face and inner face when accessible)				X	X	MAG	
	Additional where e > 50mm				X	X	RAD	
	Additional where e > 50mm (* as per provisions in RS 7721.2)				X	X	US US(*)	
3. Full penetration angle joint (Types III.1, III.2)	All thicknesses				X	X	RES	RS 7724.1 RS 7724.2 RS 7724.3
					X	X	MAG	
					X	X	RAD (*)	



(\*) If RAD cannot be performed, carry out a US examination (see **RMC 2610** for austenitic steels)  
If neither RAD nor US can be performed and more particularly for assemblies of flanges and nozzles of internal diameter not exceeding 60 mm see examination during weld operation.

# Conclusions

## Main remarks

- The on going activities for the monolith vessel are on schedule for Lower and Medium vessel
- Mechanical analysis shows room for optimization of thicknesses.
- The simplified model for the target has been accepted by ESS for seismic analysis.
- Manufacturing review on going without considering vacuum manufacturing procedures.